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## Distributed Loads on Elastic Foundations: The Uniform Circular Load

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FORMULAS are presented for computing the stresses, strains, and deflections produced at a point within a semi-infinite, homogeneous, elastically isotropic body by a uniform circular load applied to its surface. The expressions presented are obtained principally by extending and developing the work of A. E. H. Love. Reference is made also to an earlier work by Kwan-ichi Terazawa who obtained a solution by a method different from Love's. Some discussion is also included on the relation between the two methods. It is suggested that the formulas may be useful in developing theoretical concepts relating to the design of airfield pavements.

● THIS paper presents the results of certain mathematical studies (1) made in connection with projects for improving the present methods for the design of airfield pavements. In computing the theoretical values of the stresses, strains, and deflections produced in the pavement or subgrade by the loads on the airplane wheels, it was found convenient to assume the tire contact area as circular and to employ formulas for a uniform circular load

from the theory of elasticity. Where the point at which such values are to be computed is taken directly beneath the center of the circular area, the computations may be made from relatively simple formulas. However, for points not under the center, the formulas are much more complex and in some cases had not been completely developed.

Values at such "offset" points can be obtained graphically by means of charts de-

veloped by Professor N. M. Newmark (4, 5) but to obtain a more complete understanding of the pattern of stress distribution beneath the plane's wheels, it was considered desirable to employ mathematical functions. The most complete treatment of the uniform circular load is that of A. E. H. Love (2) and it is with his work that this paper largely concerns itself. Some reference will, however, also be made to an earlier work by Kwan-ichi Terazawa (6).

Love's paper is well known and has been widely quoted in the various treatises dealing with the theory of elasticity, but the formulas presented therein are too complex for computational use by the average engineer and, in some cases, are not completely developed. So far as is known, no statement of Love's findings has heretofore been presented in more detailed form. Love presents stress formulas in the form of partial derivatives using cylindrical coordinates for the case of the uniform circular load on a semi-infinite, homogeneous, isotropic, elastic body. These are reproduced here in column B of table 1.

In his discussion of the uniform circular load, Love refers to paragraph 1, page 379, where formulas are presented for the general case of a uniform load distributed over any area. These formulas are expressed in rectangular coordinates and are reproduced here in column A of table 1. Love expresses the various stresses and deflections in terms of the first and second partial derivatives of  $\chi$  (the Boussinesq three-dimensional logarithmic potential) and  $V$  (Newton's potential of a surface distribution).

These two functions are given by the double integrals

$$\chi = p \iint \log [z + \sqrt{(x-x')^2 + (y-y')^2 + z^2}] dx' dy'$$

$$V = p \iint \frac{dx' dy'}{\sqrt{(x-x')^2 + (y-y')^2 + z^2}}$$

where  $x, y,$  and  $z$  are taken as the coordinates of the point within the solid at which the stress or deflection is to be expressed and  $x', y',$  and  $0$  as the coordinates of the point on the surface at which the elemental pressure  $p dx' dy'$  is applied.

It can be shown by differentiating under the

integral sign that  $V$  equals  $\partial\chi/\partial z$ . Values for the first and second partial derivatives of  $\chi$  and  $V$  with respect to  $\rho(\partial\chi/\partial\rho, \partial^2\chi/\partial\rho^2, \partial V/\partial\rho,$  and  $\partial^2V/\partial\rho^2)$  can be obtained by applying Green's theorem, (7, pages 191-192) integrating by parts, and substituting an identity for the value of the trigonometric function of the numerator.

The resulting parts are integrated separately with some of the terms being the standard elliptic integral forms. Values for the first and second derivatives of  $V$  with respect to  $z(\partial V/\partial z$  and  $\partial^2V/\partial z^2)$  can be found from the values of the derivatives of  $\chi$  and  $V$  with respect to  $\rho$  in conjunction with the relations:

$$\nabla^2 \chi = 0; \quad \nabla^2 V = 0; \quad \frac{\partial\chi}{\partial z} = V$$

where  $\nabla^2$  denotes the operation:

$$\frac{\partial^2}{\partial\rho^2} + \frac{1}{\rho} \frac{\partial}{\partial\rho} + \frac{\partial^2}{\partial z^2}$$

The methods employed by Love in evaluating the preceding derivatives of  $\chi$  and  $V$  with respect to  $\rho$  and  $z$  are not adaptable to the integration of  $\partial\chi/\partial z$  and this step had to be accomplished by other means. Love states on page 398 that "an expression for  $V$  as an elliptic integral is known"; however, he does not present it in his paper. A value for  $\partial V/\partial z$  as a single integral, obtained from the relation  $\nabla^2 = 0$ , was integrated to give the value for  $V$ . This is shown on line 23, table 2. This value was later confirmed by transformations from Terazawa's terms.

It will be noted from the terms in columns A and B of table 1 that Love employs Lamé's elastic constants  $\lambda$  and  $\mu$ . These have been translated to terms containing the more familiar Poisson's ratio  $\nu$  and modulus of elasticity ( $E_m$ ). The translated terms are shown in column C of table 1 and the substitutions made in each case are shown in the notations listed in table 4.

The derivative terms required for the expression of the various stresses, strains, and deflections, are presented in column D of table 2. This is simply a convenient tabulation of the terms employed by Love. In columns E, F, G, and H of table 2, the values of the derivatives are given in terms of elliptic integrals for four cases:  $\rho$  less than  $r, \rho$  more than  $r, \rho$  equal to  $r,$  and  $\rho$  equal to 0. The numbers

TABLE 1  
BASIC DIFFERENTIAL EQUATIONS FROM THE WORK OF A. E. H. LOVE

Line Number	Love's Terms	COLUMN A Load distributed over any area Love page 377, formulas 1.1 (5 and 7)	Love's Terms	COLUMN B Load distributed over a circular area Love page 390, formulas 3.1 (4 and 5)	WFS Terms	COLUMN C Terms obtained from those shown in Column B by substituting constants of elasticity shown on Table 4
1	$\bar{x}_x$	$\frac{1}{2\pi} \left[ \frac{\lambda}{\lambda + \mu} \frac{\partial v}{\partial z} - \frac{\mu}{\lambda + \mu} \frac{\partial^2 x}{\partial x^2} - z \frac{\partial^2 v}{\partial x^2} \right]$	$\bar{\rho}\bar{\rho}$	$\frac{1}{2\pi} \left[ \frac{\lambda}{\lambda + \mu} \frac{\partial v}{\partial z} - \frac{\mu}{\lambda + \mu} \frac{\partial^2 x}{\partial \rho^2} - z \frac{\partial^2 v}{\partial \rho^2} \right]$	$\sigma_\rho$	$\frac{1}{2\pi} \left[ \frac{\partial v}{\partial z} - (1 - 2\nu) \frac{\partial^2 x}{\partial \rho^2} - z \frac{\partial^2 v}{\partial \rho^2} \right]$
2	$\bar{y}\bar{y}$	$\frac{1}{2\pi} \left[ \frac{\lambda}{\lambda + \mu} \frac{\partial v}{\partial z} - \frac{\mu}{\lambda + \mu} \frac{\partial^2 x}{\partial y^2} - z \frac{\partial^2 v}{\partial y^2} \right]$	$\bar{\omega}\bar{\omega}$	$\frac{1}{2\pi} \left[ \frac{\lambda}{\lambda + \mu} \frac{\partial v}{\partial z} - \frac{\mu}{\lambda + \mu} \frac{1}{\rho} \frac{\partial x}{\partial \rho} - z \frac{1}{\rho} \frac{\partial v}{\partial \rho} \right]$	$\sigma_\theta$	$\frac{1}{2\pi} \left[ \frac{\partial v}{\partial z} - (1 - 2\nu) \frac{1}{\rho} \frac{\partial x}{\partial \rho} - z \frac{1}{\rho} \frac{\partial v}{\partial \rho} \right]$
3	$\bar{z}\bar{z}$	$\frac{1}{2\pi} \left[ \frac{\partial v}{\partial z} - z \frac{\partial^2 v}{\partial z^2} \right]$	$\bar{z}\bar{z}$	$\frac{1}{2\pi} \left[ \frac{\partial v}{\partial z} - z \frac{\partial^2 v}{\partial z^2} \right]$	$\sigma_z$	$\frac{1}{2\pi} \left[ \frac{\partial v}{\partial z} - z \frac{\partial^2 v}{\partial z^2} \right]$
4	$\bar{z}\bar{x}$	$-\frac{1}{2\pi} z \frac{\partial^2 v}{\partial z \partial x}$	$\bar{\rho}\bar{z}$	$-\frac{z}{2\pi} \frac{\partial^2 v}{\partial \rho \partial z}$	$\tau_{\rho z}$	$-\frac{z}{2\pi} \frac{\partial^2 v}{\partial \rho \partial z}$
5	$u$	$-\frac{1}{4\pi} \left[ \frac{1}{\lambda + \mu} \frac{\partial x}{\partial x} + \frac{z}{\mu} \frac{\partial v}{\partial x} \right]$	$u_\rho$	$\frac{1}{4\pi} \left[ \frac{1}{\lambda + \mu} \frac{\partial x}{\partial \rho} + \frac{z}{\mu} \frac{\partial v}{\partial \rho} \right]$	$w_\rho$	$-\frac{(1 + \nu)}{2\pi k_m} \left[ (1 - 2\nu) \frac{\partial x}{\partial \rho} + z \frac{\partial v}{\partial \rho} \right]$
6	$v$	$-\frac{1}{4\pi} \left[ \frac{1}{\lambda + \mu} \frac{\partial x}{\partial y} + \frac{z}{\mu} \frac{\partial v}{\partial y} \right]$			$w_\theta$	0
7	$w$	$\frac{1}{4\pi\mu} \left[ \frac{\lambda + 2\mu}{\lambda + \mu} v - z \frac{\partial v}{\partial z} \right]$	$u_z$	$\frac{1}{4\pi\mu} \left[ \frac{\lambda + 2\mu}{\lambda + \mu} v - z \frac{\partial v}{\partial z} \right]$	$w_z$	$+\frac{(1 + \nu)}{2\pi k_m} \left[ 2(1 - \nu)v - z \frac{\partial v}{\partial z} \right]$

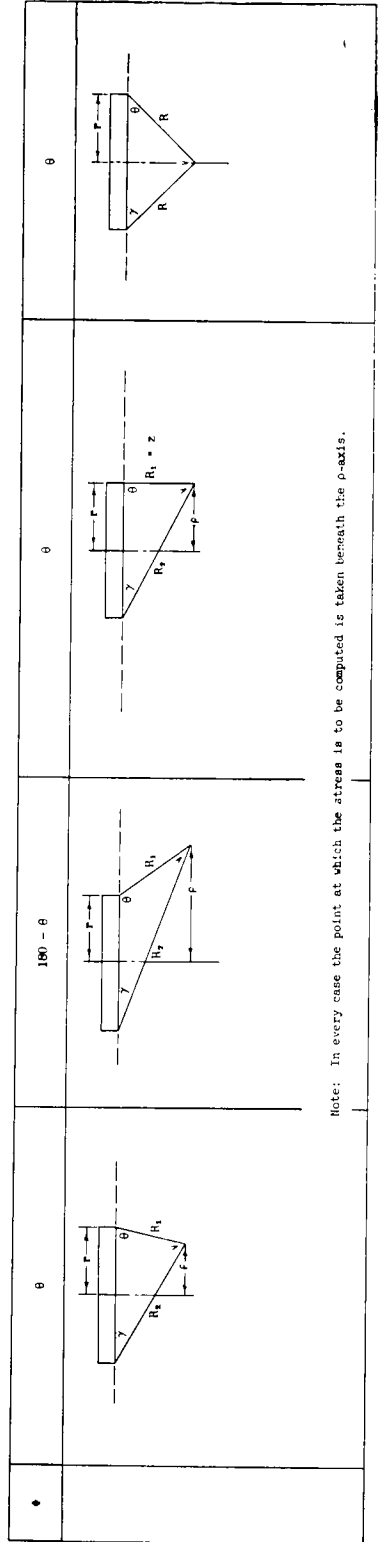
ADDITIONAL TERMS

Line Number	COLUMN A		COLUMN B		COLUMN C	
	MES Terms	Formulas from theory of elasticity (Timoshenko) and integrals for force summations	Terms obtained from Column A by substituting constants of elasticity shown on Table 4 and terms of Column A	Terms obtained from Column A or B <sub>1</sub> (v = 0.3)		
8	$\epsilon_p$	$\frac{\sigma_p - v(\sigma_\theta + \sigma_z)}{E_m}$	$-\frac{(1+v)}{2\pi E_m} \left[ (1-2\nu) \frac{\partial^2 X}{\partial \rho^2} + z \frac{\partial^2 V}{\partial \rho^2} \right]$	$-\frac{3z}{4\pi E_m} \frac{\partial^2 V}{\partial \rho^2}$		
9	$\epsilon_\theta$	$\frac{\sigma_\theta - v(\sigma_p + \sigma_z)}{E_m}$	$-\frac{(1+v)}{2\pi E_m} \left[ (1-2\nu) \frac{1}{\rho} \frac{\partial X}{\partial \rho} + z \frac{1}{\rho} \frac{\partial V}{\partial \rho} \right]$	$-\frac{3z}{4\pi E_m} \frac{1}{\rho} \frac{\partial V}{\partial \rho}$		
10	$\epsilon_z$	$\frac{\sigma_z - v(\sigma_p + \sigma_\theta)}{E_m}$	$\frac{(1+v)}{2\pi E_m} \left[ (1-2\nu) \frac{\partial V}{\partial z} - z \frac{\partial^2 V}{\partial z^2} \right]$	$-\frac{3z}{4\pi E_m} \frac{\partial^2 V}{\partial z^2}$		
11	$e$	$\epsilon_p + \epsilon_\theta + \epsilon_z$	$\frac{(1+v)(1-2\nu)}{\pi E_m} \frac{\partial V}{\partial z}$	0		
12	$\phi = \sigma_{\theta\theta}$	$\sigma_p + \sigma_\theta + \sigma_z$	$\frac{1}{\pi} (1+v) \frac{\partial V}{\partial z}$	$\frac{3}{2\pi} \frac{\partial V}{\partial z}$		
13	$2\pi \int_0^{\rho_1} \rho \sigma_z d\rho$	$\int_0^{\rho_1} \rho \left[ \frac{\partial V}{\partial z} - z \frac{\partial^2 V}{\partial z^2} \right] d\rho$		$\left[ \rho z \frac{\partial V}{\partial \rho} - \rho \frac{\partial X}{\partial \rho} \right]_0^{\rho_1}$		
14	$2\pi \int_{z_1}^{z_2} \rho \tau_{pz} dz$	$-\rho \int_{z_1}^{z_2} z \frac{\partial^2 V}{\partial \rho \partial z} dz$		$\left[ -\rho z \frac{\partial V}{\partial \rho} + \rho \frac{\partial X}{\partial \rho} \right]_{z_1}^{z_2}$		
15	$\int_0^{\rho_1} \tau_{z\rho} d\rho$	$-\frac{z}{2\pi} \int_0^{\rho_1} \frac{\partial^2 V}{\partial \rho \partial z} d\rho$		$-\frac{z}{2\pi} \left[ \frac{\partial V}{\partial z} \right]_0^{\rho_1}$		

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TABLE 2  
EVALUATION OF DERIVATIVES

Line Number	COLUMN D		COLUMN E		COLUMN F		COLUMN G		COLUMN H	
	Derivatives	Love's Formula No.	Case I $\rho < r$	Love's Formula No.	Case II $\rho > r$	Case III $\rho = r$	Case IV $\rho = 0$			
16	$\frac{1}{p} \frac{\partial x}{\partial \rho}$	3.24-1	$p \left[ \pi + \frac{r^2 - \rho^2}{\rho^2} J + \frac{z}{\rho^2 R_1} \left\{ R_1^2 E' - (2R_1 R_2 b - z^2) K' \right\} \right]$	3.24-2	$p \left[ \frac{\pi r^2 - r^2 - \rho^2}{\rho^2} J + \frac{z}{\rho R_1} \left\{ R_1^2 E' - (2R_1 R_2 b - z^2) K' \right\} \right]$	$p \left[ \pi + \frac{2R_1^2}{\rho^2} E' - \frac{2R_2}{\rho^2} K' \right]$	$p\pi \left[ 1 - \frac{z}{R_1} \right]$			
17	$\frac{\partial^2}{\partial \rho^2}$	3.3-6	$p \left[ \pi - \frac{r^2 + \rho^2}{\rho^2} J - \frac{2R_1}{\rho^2} E' + \frac{z(2r^2 + z^2)}{R_1 \rho^2} K' \right]$	3.3-6	$p \left[ \frac{\pi r^2 + r^2 + \rho^2}{\rho^2} J - \frac{2R_1}{\rho^2} E' + \frac{z(2r^2 + z^2)}{R_1 \rho^2} K' \right]$	$\frac{2p}{R_1 \rho^2} \left[ (2r^2 + z^2) K' - R_1^2 E' \right]$	$p\pi \left[ 1 - \frac{z}{R_1} \right]$			
18	$\frac{\partial v}{\partial z} = - \left( \frac{1}{\rho} \frac{\partial x}{\partial \rho} + \frac{\partial^2}{\partial \rho^2} \right)$	3.4-1	$-2p \left[ \pi - J - \frac{z}{R_1} K' \right]$	3.4-2	$-2p \left[ J - \frac{z}{R_1} K' \right]$	$p \left[ \frac{2z}{R_1} K' - \pi \right]$	$-2p\pi \left[ 1 - \frac{z}{R_1} \right]$			
19	$\frac{\partial^2 v}{\partial z^2} = \left( \frac{1}{\rho} \frac{\partial v}{\partial \rho} + \frac{\partial^2 v}{\partial \rho^2} \right)$	3.5-4	$2p \left[ \frac{K'}{R_1} - \frac{\cos \gamma}{R_1} E' \right] + p \left[ \frac{2K'}{R_1} - \frac{R_2}{\rho^2} \left\{ 1 - ab \right\} E' \right]$	Same as Case I.	Same as Case I.	$p \left[ \frac{2}{R_1} K' - \frac{2}{R_1} E' \right]$	$2p\pi \frac{R_2}{R_1}$			
20	$\frac{1}{p} \frac{\partial v}{\partial \rho}$	3.5-1	$-p \frac{R_2}{\rho} \left[ (1 + k^2) K' - 2E' \right] = -p \frac{2}{\rho} \left\{ R_1 K' - R_1^2 E' \right\}$	Same as Case I.	Same as Case I.	$-p \frac{2}{\rho^2 R_1} \left[ (2r^2 + z^2) K' - R_1^2 E' \right]$	$-p\pi \frac{R_2}{R_1}$			
21	$\frac{\partial^2 v}{\partial \rho^2}$	3.5-2	$\frac{p}{\rho R_1} \left[ 2(r^2 + z^2) K' - R_1^2 (1 + ab) E' \right]$	Same as Case I.	Same as Case I.	$p \left[ \frac{2(z^2 + r^2) K'}{\rho^2 R_1} - \frac{6r^2 + 2z^2}{R_1 \rho^2} E' \right]$	$-p\pi \frac{R_2}{R_1}$			
22	$\frac{\partial^2 v}{\partial \rho \partial z}$	3.5-3	$\frac{z}{\rho R_1} \left[ \left( 1 + \frac{1}{k^2} \right) (E' - 2K') \right]$	Same as Case I.	Same as Case I.	$\frac{2p}{2R_1 \rho} \left[ (2r^2 + z^2) E' - z^2 K' \right]$	0			
23	$v \cdot \frac{\partial v}{\partial z}$		$-2p \left[ z(n - J) - \frac{1}{R_1} \left\{ R_1^2 E' + (r^2 - \rho^2) K' \right\} \right]$			$p \left[ -nz + 2R_1 K' \right]$	$-2p\pi(z - R_1)$			



Note: In every case the point at which the stress is to be computed is taken beneath the  $\rho$ -axis.

to the left of columns E and F refer to the formula numbers in Love's paper.

Formulas for the stresses and deflections have been obtained by substituting the values for the various derivatives shown in columns E, F, G, and H of table 2 in the expressions shown in column C of table 1. Additional formulas have also been developed for the strains and for certain summations and are shown in column C<sub>1</sub> of table 1. These formulas (shown in table 3) are given only for a value of Poisson's ratio of 0.5. This value assumes that the stressed elastic body has undergone no change in volume and was selected because it yields the simplest expressions. However, formulas for other values of Poisson's ratio can be obtained by making suitable substitutions in the expressions shown in table 1.

Love also shows that all the stresses produced by a uniform circular load at a point Q may be expressed in terms of the solid angle subtended at Q by the circular area. The expressions which Love gives for the solid angle (page 397) can be obtained from the expressions for the partial derivative of V with respect to z(∂V/∂z). This expression is shown on line 18 of table 2.

Several methods were employed in checking the accuracy of the final expressions. Simplest of these is that of checking the previously developed formulas for special cases. Where the point is taken beneath the center of the circular area (ρ = 0), the formulas for σ<sub>z</sub> and for σ<sub>ρ</sub> = σ<sub>θ</sub> agree with those obtained by Love (2, page 415) and by Timoshenko (9, pages 335 and 336). These terms are listed in column M of table 3.

It can also be shown that where the point is taken beneath the edge of the circular area (ρ = r), the formulas yield the terms listed in column L of table 3. These were developed by the authors by integration of the Boussinesq point load equations over the area of the circle and are given in Reference 3.

A more comprehensive check of the derived formulas was gained through conversion of results published in 1916 by Terazawa (6).<sup>1</sup>

<sup>1</sup> In introducing his paper, Love states that there are two methods used in solving the problem he is presenting. One is called the "potential method" (his method); the other he calls the Bessel's function method, and is the work of Kwan-ichi Terazawa. The stress function for the potential method is

$$= \frac{z}{2\pi} x - \frac{\nu}{\pi} \int x dz$$

Terazawa gives a solution to the problem of a uniformly loaded circular area on the boundary of a semi-infinite body by employing Bessel's functions. His results have been converted to expressions involving elliptic functions and as such are found to agree with those developed from Love's work.

Terazawa also developed the Newtonian potential of a surface distribution as an elliptic integral. This function appears in the formula for the vertical deflection. By using the transformations described in the historical note shown below, the Newtonian potential was expressed in terms which agree with those derived by the authors in interpreting Love's paper. It can also be shown (see figure 1) that the formulas yield numerical results which agree with those obtained from the Newmark charts.

In presenting the formulas for direct computation of the stresses, strains, and deflections produced by a uniform circular load, it is intended to emphasize the usefulness of such formulas for analytical purposes rather than to suggest that the values obtained by their use are more accurate or can be computed more quickly than those obtained graphically from charts. Charts for such computations can be constructed to any desired degree of accuracy and any saving of time is open to question. However, as is shown in the following paragraphs, a number of analytical studies can be accomplished with the formulas which would be impossible without them.

*Principal stresses and angles of principal planes.* Formulas for the direct computation of the principal stresses and of the angles of the principal planes may be developed by combining the formulas for the normal and shearing stresses. The general three-dimensional

while that for the Bessel's function method applied to the case of a uniform circular load is

$$\phi = \frac{2pV}{\pi r} \int_0^\infty e^{-kz} J_1(k, r) J_0(k, \rho) \frac{dk}{k^2} - \frac{zp}{\pi a}$$

$$\int_0^\infty e^{-kz} J_1(k, r) J_0(k, \rho) \frac{dk}{k^2}$$

By using methods developed by H. Nagaoko (Phil. Mag. VI: 6:1903), Terazawa developed functions identical with those obtained by Love after Love had applied Green's theorem. From this point Love and Terazawa deviate: Love develops the terms in the elliptic functions K', E', and H (φ, n, k); Terazawa develops the terms in Weierstrass and theta functions. The works of Whittaker and Watson (10) were used to develop transformations which express Terazawa's results in the terms employed by Love. These results agree in detail with those of Love.

TABLE 3  
FORMULAS: POISSON'S RATIO,  $\nu = 0.5$

Line Number	COLUMN L		COLUMN M		Deflection Strain or Integral	COLUMN L	COLUMN M
	Case I ( $\rho < r$ ), II( $\rho > r$ ), III( $\rho = r$ )	Case IV ( $\rho = 0$ )	Case I ( $\rho < r$ )	Case IV ( $\rho = 0$ )			
24	$B + A \left[ 2an - (1 + ab)m \right]$ (28)	$1 - \frac{\beta}{2} \cos \frac{\gamma}{2} + \frac{1}{2} \cos^2 \frac{\gamma}{2}$ (16)	$3^a$	$\frac{W_p}{P}$	$\frac{\partial z}{2\pi R_p^2 E_m} (bn - m)$	0	Case IV ( $\rho = 0$ )
25	$B - A \left[ (3b - a)n - 2m \right]$ (25)	$1 - \frac{\beta}{2} \cos \frac{\gamma}{2} + \frac{1}{2} \cos^2 \frac{\gamma}{2}$ (16)	35	$\frac{W_0}{P}$	0	0	0
26	$B - \frac{Z \cos \gamma}{nR_p} E'$ (24)	$1 - \frac{\beta}{R^2} = 1 - \cos^2 \frac{\gamma}{2}$ (15)	36	$\frac{W_z}{P}$	$\frac{\beta}{2\pi R_p^2 E_m} (a - n \cos \gamma)$ (35)	$\frac{\beta}{4\pi R_p^2 E_m} [(a + b)n - (1 + ab)m]$ (35)	$\frac{3r^2}{2\pi R_p^2 E_m}$ (33)
27	$3 \left[ B - \frac{Z}{R_p^2} K' \right]$ (27)	$3 \left[ 1 - \frac{Z}{R} \right]$ (17)	37	$\frac{\epsilon_p}{P}$	$\frac{\partial z}{2\pi R_p^2 E_m} (a - bn)$	$\frac{\partial z}{4\pi R_p^2 E_m} [(a + b)n - (1 + ab)m]$ (35)	$-\frac{3zr^2}{4R^2 E_m}$
28	$B - A \left[ (1 - a)n + (1 - b)m \right]$ (30)	$1 - \cos^2 \frac{\gamma}{2}$ (15)	38	$\frac{r_n}{P}$	$\frac{\partial z}{2\pi R_p^2 E_m} (a - bn)$	$\frac{\partial z}{2\pi R_p^2 E_m} (a - bn)$	$-\frac{3zr^2}{4R^2 E_m}$
29	$B - A \left[ (1 + b)m - (1 + a)n \right]$ (24)	$1 - \frac{\beta}{2} \cos \frac{\gamma}{2} + \frac{1}{2} \cos^2 \frac{\gamma}{2}$ (16)	39	$\frac{\epsilon_z}{P}$	$\frac{\partial z}{2\pi R_p^2 E_m} (n - m \cos \gamma)$ (36)	$\frac{\partial z}{2\pi R_p^2 E_m} (n - m \cos \gamma)$ (36)	$\frac{3zr^2}{2R^2 E_m}$ (34)
30	$A (bm - n)$ (32)	$\frac{\beta \cos \frac{\gamma}{2} \sin^2 \frac{\gamma}{2}}{4}$ (19)	40	$\int_0^{z_1} \tau_{z\epsilon} dz$	$2p \left[ -B + \frac{ZK'}{R_p n} + (1 - \frac{Z}{R}) \right] (\rho = \infty) zp \left[ 1 - \frac{Z}{R} \right]$	$2p \left[ -B + \frac{ZK'}{R_p n} + (1 - \frac{Z}{R}) \right] (\rho = \infty) zp \left[ 1 - \frac{Z}{R} \right]$	0
31	$\frac{Z^2}{\pi R_p R_p}$ (bm - n) (29)	0	41	$\int_0^{z_1} \sigma_p dz$	$\frac{p}{2} \left[ \frac{r^2 + p^2 -  r^2 - p^2 }{2} + \frac{ r^2 - p^2 }{\pi} \right] J + \frac{Z}{\pi R_p} (\alpha^2 K' - R_p^2 E')$	$\frac{p}{2} \left[ \frac{r^2 + p^2 -  r^2 - p^2 }{2} + \frac{ r^2 - p^2 }{\pi} \right] J + \frac{Z}{\pi R_p} (\alpha^2 K' - R_p^2 E')$	0
32	$\frac{2Z\epsilon}{R_p R_p}$	0	42	$\int_{z_1}^{z_2} \tau_{p\sigma} dz$	$\frac{p}{2} \left[ \frac{r^2 + p^2 -  r^2 - p^2 }{2} + \frac{ r^2 - p^2 }{\pi} \right] J + \frac{Z}{\pi R_p} (\alpha^2 K' - R_p^2 E')$	$\frac{p}{2} \left[ \frac{r^2 + p^2 -  r^2 - p^2 }{2} + \frac{ r^2 - p^2 }{\pi} \right] J + \frac{Z}{\pi R_p} (\alpha^2 K' - R_p^2 E')$	0
33	$180^\circ - \left[ \frac{B - 7}{2} \right]$	180°					

Note: The numbers shown in parentheses at the right of Columns L and M are the numbers of formulas given on pages A10 through A19 of reference 3.

TABLE 4  
NOTATIONS

Relations between Love's Terms and Those Used by WES		
Love's Terms	WES Terms	WES Terms
Rectangular Coordinates	Cylindrical Coordinates	Spherical Coordinates
x	$\rho \cos \omega$	$r$
y	$\rho \sin \omega$	$\theta$
z	z	z
x'	$r' \cos \omega'$	$r \cos \theta_1$
y'	$r' \sin \omega'$	$r \sin \theta_1$
$r_1$	$r_1$	$R_1$
$r_2$	$r_2$	$R_2$
-	a	r
$\frac{\partial^2 V}{\partial x^2}$	$\frac{\partial^2 V}{\partial \rho^2}$	$\frac{\partial^2 V}{\partial r^2}$
$\frac{\partial^2 \chi}{\partial x^2}$	$\frac{\partial^2 \chi}{\partial \rho^2}$	$\frac{\partial^2 \chi}{\partial r^2}$
$\frac{\partial^2 V}{\partial y^2}$	$\frac{1}{\rho} \frac{\partial V}{\partial \rho}$	$\frac{1}{r} \frac{\partial V}{\partial r}$
$\frac{\partial^2 \chi}{\partial y^2}$	$\frac{1}{\rho} \frac{\partial \chi}{\partial \rho}$	$\frac{1}{r} \frac{\partial \chi}{\partial r}$

Constants of Theory of Elasticity	
Lamé's Terms Used by Love	WES Terms
$\frac{\lambda}{2(\lambda + \mu)}$	$\nu$
$\frac{\mu}{\lambda + \mu}$	$1 - 2\nu$
$\frac{\mu(3\lambda + 2\mu)}{\lambda + \mu}$	$E_M$
$\frac{1}{\lambda + \mu}$	$\frac{c(1 + \nu)(1 - 2\nu)}{E_M}$

Notations for Table 3	
A	$\frac{z}{2^2 \rho^2}$
B	$1 - J/\pi$
$\rho < r$	$J/\pi$
$\rho > r$	$1/2$
$\rho = r$	
m	$R_2 E^*$
n	$R_1 k^*$

Notation and Values of Factors for Certain Values of $\rho$			
Factor	General Case $z = \rho$	$\rho = r$	$\rho = 0$
H	$\sqrt{z^2 + r^2}$	H	H
$H_1$	$\sqrt{z^2 + (r - \rho)^2}$	z	H
$H_2$	$\sqrt{z^2 + (r + \rho)^2}$	$\sqrt{z^2 + 4r^2}$	H
k	$\frac{H_1}{H_2}$	$\frac{z}{\sqrt{z^2 + 4r^2}}$	1
$k^*$	$\sqrt{1 - k^2}$	$\frac{2r}{\sqrt{z^2 + 4r^2}}$	1
E	$\int_0^{\pi/2} \sqrt{1 - k^2 \sin^2 \theta} d\theta$	E	E
K	$\int_0^{\pi/2} \frac{d\theta}{\sqrt{1 - k^2 \sin^2 \theta}}$	K	$\infty$
$E^*$	$\int_0^{\pi/2} \sqrt{1 - k'^2 \sin^2 \theta} d\theta$	$E^*$	$\frac{\pi}{2}$
$K^*$	$\int_0^{\pi/2} \frac{d\theta}{\sqrt{1 - k'^2 \sin^2 \theta}}$	$K^*$	$\frac{\pi}{2}$
$E(k, \phi)$	$\int_0^{\phi} \sqrt{1 - k^2 \sin^2 \theta} d\theta$	E	$\frac{z}{H}$
$F(k, \phi)$	$\int_0^{\phi} \frac{d\theta}{\sqrt{1 - k^2 \sin^2 \theta}}$	K	$\text{arc tanh } \frac{z}{H}$
J	$K^* E(k, \phi) - (k'^2 - E^*) F(k, \phi)$	$\frac{\pi}{2}$	$\frac{\pi z}{2H}$
$\sin \phi$	$\frac{z}{H_1}$	1	$\frac{z}{H}$
$\sin \nu$	$\frac{2rz}{H_1 H_2}$	$\frac{2r}{H_2}$	$\frac{2rz}{H^2}$
$\cos \nu$	$\frac{z^2 - r^2 + \rho^2}{H_1 H_2}$	$\frac{z}{H_2}$	$\frac{z^2 - r^2}{H^2}$
$\cos \gamma$	$\frac{R_1 R_2}{2\rho^2} (1 - ab)$		
a	$\frac{z^2 + r^2 - \rho^2}{R_1 R_2}$	$\frac{z}{H_2}$	1
A	$\cos(\theta - \gamma)$		
a	$\frac{rz \cos \nu + (r^2 - \rho^2) \sin \nu}{rz}$	$\frac{z}{H_2}$	1
b	$\frac{z^2 + r^2 + \rho^2}{H_1 H_2}$	$\frac{z^2 + 2r^2}{2H_2}$	1
b	$\frac{z \cos \nu + r \sin \nu}{z}$	$\frac{z^2 + 2r^2}{2H_2}$	1

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formula for the principal stresses is given by Timoshenko (Formula 108, page 186 of reference 9) and is a cubic equation in rectangular coordinates. It is convenient to translate this expression to cylindrical coordinates ( $\rho, \theta$ , and  $z$ ), and to observe that  $\sigma_\theta$  acts on a plane of symmetry and is one of the three principal stresses.

The other two principal stresses are found from the expression:

$$S = \frac{\sigma_z + \sigma_\rho \pm \sqrt{(\sigma_z - \sigma_\rho)^2 + (2\tau_{\rho z})^2}}{2}$$

which is obtained by dividing Timoshenko's cubic equation by the factor  $(S - \sigma_\theta)$  and



TABLE 5

$$E = \int_0^\phi \sqrt{1 - k^2 \sin^2 \theta} \, d\theta \quad F = \int_0^\phi \frac{d\theta}{\sqrt{1 - k^2 \sin^2 \theta}}$$

$\phi$	$\arcsin k$	$E$	$F$	$\phi$	$\arcsin k$	$E$	$F$	$\phi$	$\arcsin k$	$E$	$F$	$\phi$	$\arcsin k$	$E$	$F$
0	0	.0000	.0000	19	0	.3316	.3316	24	70	.4082	.4301	29	30	.5010	.5114
	90	.0000	.0000		20	.3309	.3323		75	.4076	.4308		35	.4993	.5132
1	0	.0175	.0175		30	.3301	.3331		80	.4071	.4313		40	.4975	.5150
	90	.0175	.0175		40	.3291	.3341		90	.4067	.4317		45	.4957	.5170
2	0	.0349	.0349		50	.3286	.3347						50	.4938	.5190
	90	.0349	.0349		55	.3281	.3352	25	0	.4963	.4363		55	.4920	.5210
3	0	.0524	.0524		60	.3271	.3362		10	.4959	.4367		60	.4903	.5229
	90	.0524	.0524		70	.3263	.3371		15	.4954	.4372		65	.4887	.5247
4	0	.0698	.0698		80	.3256	.3379		20	.4948	.4379		70	.4874	.5262
	90	.0698	.0698		90				25	.4942	.4384		75	.4863	.5275
5	0	.0873	.0873		0	.3491	.3491		30	.4939	.4387		80	.4855	.5285
	90	.0872	.0874		20	.3483	.3499		35	.4930	.4397		90	.4848	.5293
6	0	.1047	.1047		30	.3473	.3508		40	.4924	.4408				
	90	.1045	.1049		35	.3468	.3514		45	.4918	.4420	30	0	.5236	.5236
7	0	.1222	.1222		40	.3462	.3520		50	.4912	.4433		10	.5229	.5243
	90	.1219	.1225		45	.3456	.3526		55	.4907	.4446		15	.5221	.5251
8	0	.1396	.1396		50	.3450	.3533		60	.4901	.4458		20	.5213	.5263
	90	.1392	.1401		55	.3444	.3539		65	.4896	.4470		25	.5205	.5277
9	0	.1571	.1571		60	.3438	.3545		70	.4890	.4481		30	.5197	.5294
	90	.1564	.1577		70	.3429	.3555		75	.4884	.4493		35	.5189	.5313
10	0	.1745	.1745		80	.3420	.3564		80	.4878	.4504		40	.5181	.5334
	90	.1736	.1754		90				85	.4872	.4515		45	.5172	.5356
11	0	.1920	.1920		0	.3665	.3665		90	.4866	.4526		50	.5164	.5379
	45	.1914	.1926		20	.3656	.3675						55	.5156	.5401
	90	.1908	.1932		25	.3651	.3680	26	0	.4538	.4538		60	.5148	.5422
12	0	.2269	.2269		30	.3645	.3685		10	.4533	.4542		65	.5140	.5442
	45	.2087	.2102		35	.3639	.3692		15	.4528	.4548		70	.5132	.5459
	90	.2079	.2110		40	.3632	.3699		20	.4522	.4556		75	.5124	.5474
13	0	.2269	.2269		45	.3625	.3706		25	.4517	.4565		80	.5116	.5484
	45	.2259	.2279		50	.3618	.3714		30	.4511	.4576		85	.5108	.5493
	90	.2250	.2289		55	.3611	.3728		35	.4506	.4588		90	.5100	.5516
14	0	.2443	.2443		60	.3604	.3743		40	.4500	.4602				
	30	.2437	.2450		65	.3598	.3754		45	.4495	.4616	31	0	.5411	.5411
	45	.2431	.2456		70	.3592	.3767		50	.4489	.4630		10	.5403	.5418
	60	.2425	.2462		75	.3586	.3777		55	.4484	.4645		15	.5394	.5427
	90	.2419	.2468		80	.3580	.3787		60	.4478	.4658		20	.5386	.5440
15	0	.2618	.2618		90	.3574	.3798		65	.4473	.4670		25	.5378	.5456
	30	.2611	.2625						70	.4467	.4681		30	.5370	.5475
	45	.2603	.2633		0	.3840	.3840		75	.4462	.4693		35	.5362	.5491
	60	.2596	.2641		10	.3837	.3842		80	.4457	.4704		40	.5354	.5509
	90	.2588	.2648		20	.3829	.3851		85	.4452	.4715		45	.5346	.5523
16	0	.2793	.2793		25	.3823	.3856		90	.4447	.4726		50	.5338	.5538
	30	.2784	.2802		30	.3817	.3863						55	.5330	.5553
	45	.2775	.2811		35	.3809	.3871	27	0	.4712	.4712		60	.5322	.5568
	60	.2765	.2820		40	.3802	.3879		10	.4707	.4717		65	.5314	.5583
	90	.2756	.2830		45	.3793	.3887		15	.4701	.4724		70	.5306	.5598
17	0	.2967	.2967		50	.3785	.3896		20	.4696	.4732		75	.5298	.5613
	20	.2962	.2972		55	.3777	.3904		25	.4690	.4743		80	.5290	.5628
	30	.2956	.2978		60	.3770	.3912		30	.4685	.4755		85	.5282	.5643
	40	.2949	.2985		65	.3763	.3919		35	.4680	.4769		90	.5274	.5658
	45	.2942	.2989		70	.3757	.3926		40	.4675	.4784				
	50	.2935	.2993		75	.3749	.3935		45	.4670	.4799	32	0	.5585	.5585
	60	.2929	.3006		80	.3742	.3943		50	.4665	.4813		10	.5577	.5593
	90	.2924	.3012		90	.3736	.3952		55	.4660	.4828		15	.5569	.5603
18	0	.3142	.3142						60	.4655	.4843		20	.5561	.5617
	20	.3136	.3148		0	.4014	.4014		65	.4650	.4858		25	.5553	.5632
	30	.3129	.3154		10	.4011	.4017		70	.4645	.4873		30	.5545	.5656
	40	.3121	.3163		20	.4002	.4027		75	.4640	.4884		35	.5537	.5677
	45	.3116	.3167		25	.3996	.4033		80	.4635	.4891		40	.5529	.5698
	50	.3112	.3172		30	.3988	.4041		85	.4630	.4897		45	.5521	.5714
	60	.3103	.3181		35	.3980	.4049		90	.4625	.4907		50	.5513	.5735
	70	.3096	.3188		40	.3971	.4059						55	.5505	.5756
	90	.3090	.3195		45	.3961	.4068	28	0	.4887	.4887		60	.5497	.5777
					50	.3952	.4078		10	.4881	.4893		65	.5489	.5798
					55	.3943	.4088		15	.4876	.4899		70	.5481	.5819
					60	.3935	.4097		20	.4870	.4909		75	.5473	.5840
					65	.3927	.4105		25	.4865	.4921		80	.5465	.5861
					70	.3920	.4113		30	.4860	.4934		85	.5457	.5882
					75	.3911	.4123		35	.4855	.4950		90	.5449	.5903
					80	.3902	.4132		40	.4850	.4967				
					85	.3893	.4141		45	.4845	.4985	33	0	.5760	.5760
					90	.3884	.4150		50	.4840	.4999		10	.5752	.5769
									55	.4835	.5013		15	.5744	.5780
									60	.4830	.5021		20	.5736	.5795
									65	.4825	.5034		25	.5728	.5816
									70	.4820	.5053		30	.5720	.5837
									75	.4815	.5067		35	.5712	.5858
									80	.4810	.5079		40	.5704	.5879
									85	.4805	.5097		45	.5696	.5900
									90	.4800	.5099		50	.5688	.5921
													55	.5680	.5942
													60	.5672	.5963
													65	.5664	.5984
													70	.5656	.6005
													75	.5648	.6026
													80	.5640	.6047
													85	.5632	.6068
													90	.5624	.6089

TABLE 5—Continued

$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$	
33	80	.5456	.6095	38	40	.6444	.6831	43	0	.7505	.7505	47	55	.7628	.8860	
	90	.5446	.6107		45	.6403	.6877		10	.7486	.7524		60	.7555	.8958	
34	0	.5934	.5934	50	.6361	.6925	15	.7463	.7548	65	.7488	.9053				
	10	.5924	.5944	55	.6321	.6973	20	.7431	.7550	70	.7429	.9139				
	15	.5912	.5956	60	.6282	.7019	25	.7391	.7622	75	.7380	.9212				
	20	.5896	.5973	65	.6247	.7063	30	.7345	.7671	80	.7344	.9269				
	25	.5876	.5994	70	.6216	.7102	35	.7293	.7728	90	.7314	.9316				
	30	.5852	.6018	75	.6191	.7135	40	.7237	.7791	48	0	.8378	.8378			
	35	.5826	.6046	80	.6172	.7159	45	.7178	.7859		10	.8352	.8403			
	40	.5797	.6077	90	.6157	.7180	50	.7118	.7931		15	.8320	.8436			
	45	.5768	.6109	39	0	.6807	.6807	55	.7059		.8004	20	.8276	.8480		
	50	.5738	.6143		10	.6792	.6821	60	.7003		.8075	25	.8223	.8537		
55	.5709	.6176	15		.6775	.6839	65	.6952	.8143		30	.8160	.8606			
60	.5681	.6208	20		.6750	.6864	70	.6907	.8204		35	.8089	.8685			
65	.5656	.6238	25		.6720	.6895	75	.6870	.8256		40	.8012	.8773			
70	.5634	.6265	30		.6685	.6932	80	.6843	.8295		45	.7931	.8870			
75	.5616	.6287	35		.6646	.6975	90	.6820	.8328		50	.7849	.8973			
80	.5603	.6303	40		.6604	.7021	44	0	.7679	.7679	55	.7768	.9079			
90	.5592	.6317	45		.6559	.7071		10	.7659	.7700	60	.7690	.9185			
35	0	.6109	.6109		50	.6515		.7123	15	.7634	.7725	65	.7618	.9287		
	10	.6098	.6119	55	.6471	.7176		20	.7600	.7760	70	.7555	.9381			
	15	.6085	.6133	60	.6429	.7227		25	.7558	.7804	75	.7502	.9461			
	20	.6067	.6151	65	.6391	.7275		30	.7508	.7857	80	.7464	.9523			
	25	.6045	.6173	70	.6357	.7318		35	.7453	.7918	90	.7431	.9575			
	30	.6019	.6200	75	.6330	.7353		40	.7393	.7986	49	0	.8552	.8552		
	35	.5991	.6231	80	.6310	.7380		45	.7330	.8059		10	.8525	.8579		
	40	.5960	.6264	90	.6293	.7403		50	.7266	.8136		15	.8491	.8614		
	45	.5928	.6300	40	0	.6981	.6981	55	.7204	.8215		20	.8446	.8661		
	50	.5895	.6336		10	.6966	.6997	60	.7144	.8293		25	.8389	.8721		
55	.5863	.6373	15		.6947	.7016	65	.7088	.8367	30		.8322	.8794			
60	.5833	.6408	20		.6921	.7043	70	.7040	.8433	35		.8247	.8878			
65	.5806	.6441	25		.6888	.7076	75	.7000	.8490	40		.8165	.8972			
70	.5782	.6471	30		.6851	.7116	80	.6971	.8533	45		.8079	.9076			
75	.5762	.6495	35		.6808	.7162	90	.6947	.8569	50		.7992	.9186			
80	.5748	.6513	40		.6763	.7213	45	0	.7854	.7854	55	.7905	.9300			
90	.5736	.6528	45		.6715	.7267		10	.7832	.7876	60	.7822	.9415			
36	0	.6283	.6283		50	.6667		.7323	15	.7806	.7903	65	.7746	.9525		
	10	.6272	.6295	55	.6620	.7380		20	.7770	.7940	70	.7679	.9627			
	15	.6258	.6309	60	.6575	.7436		25	.7725	.7988	75	.7623	.9714			
	20	.6238	.6329	65	.6533	.7488		30	.7672	.8044	80	.7581	.9781			
	25	.6214	.6353	70	.6497	.7535		35	.7613	.8109	90	.7547	.9838			
	30	.6186	.6383	75	.6468	.7575		40	.7549	.8181	50	0	.8727	.8727		
	35	.6155	.6416	80	.6446	.7604		45	.7482	.8260		10	.8698	.8756		
	40	.6122	.6452	90	.6428	.7629		50	.7414	.8343		15	.8663	.8792		
	45	.6087	.6491	41	0	.7156	.7156	55	.7346	.8428		20	.8614	.8842		
	50	.6051	.6531		10	.7139	.7173	60	.7282	.8512		25	.8554	.8905		
55	.6017	.6571	15		.7119	.7193	65	.7223	.8592	30		.8483	.8982			
60	.5984	.6610	20		.7091	.7222	70	.7171	.8665	35		.8404	.9072			
65	.5954	.6647	25		.7056	.7258	75	.7129	.8727	40		.8317	.9173			
70	.5928	.6679	30		.7016	.7301	80	.7097	.8774	45		.8227	.9283			
75	.5907	.6706	35		.6970	.7350	90	.7071	.8814	50		.8134	.9401			
80	.5891	.6726	40		.6921	.7405	46	0	.8029	.8029	55	.8042	.9523			
90	.5878	.6743	45		.6870	.7463		10	.8006	.8052	60	.7954	.9647			
37	0	.6458	.6458		50	.6818		.7524	15	.7977	.8080	65	.7872	.9766		
	10	.6445	.6470	55	.6767	.7586		20	.7939	.8120	70	.7801	.9876			
	15	.6430	.6486	60	.6719	.7647		25	.7891	.8170	75	.7741	.9971			
	20	.6409	.6507	65	.6674	.7704		30	.7835	.8230	80	.7697	1.004			
	25	.6383	.6534	70	.6636	.7756		35	.7772	.8300	90	.7660	1.011			
	30	.6353	.6565	75	.6604	.7799		40	.7704	.8378	51	0	.8901	.8901		
	35	.6319	.6602	80	.6580	.7831		45	.7633	.8462		10	.8871	.8932		
	40	.6283	.6641	90	.6561	.7859		50	.7560	.8552		15	.8834	.8970		
	45	.6245	.6684	42	0	.7330	.7330	55	.7488	.8643		20	.8783	.9023		
	50	.6207	.6727		10	.7313	.7348	60	.7419	.8734		25	.8719	.9090		
55	.6169	.6771	15		.7291	.7370	65	.7356	.8821	30		.8644	.9172			
60	.6134	.6814	20		.7261	.7401	70	.7301	.8900	35		.8560	.9267			
65	.6101	.6854	25		.7224	.7440	75	.7255	.8968	40		.8469	.9374			
70	.6073	.6890	30		.7180	.7486	80	.7221	.9019	45		.8373	.9491			
75	.6050	.6919	35		.7132	.7539	90	.7193	.9063	50		.8275	.9617			
80	.6032	.6941	40		.7079	.7598	47	0	.8203	.8203	55	.8177	.9748			
90	.6018	.6960	45		.7024	.7661		10	.8179	.8227	60	.8084	.9881			
38	0	.6632	.6632		50	.6969		.7727	15	.8149	.8258	65	.7997	1.001		
	10	.6619	.6646	55	.6914	.7794		20	.8108	.8300	70	.7921	1.013			
	15	.6602	.6662	60	.6862	.7860		25	.8057	.8353	75	.7858	1.023			
	20	.6580	.6685	65	.6814	.7922		30	.7998	.8418	80	.7811	1.031			
	25	.6552	.6714	70	.6772	.7979		35	.7931	.8492	90	.7771	1.038			
	30	.6519	.6749	75	.6738	.8026		40	.7858	.8575	52	0	.9076	.9076		
	35	.6483	.6788	80	.6712	.8062		45	.7782	.8666		10	.9044	.9108		
				90	.6691	.8092		50	.7705	.8761						

TABLE 5—Continued

$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$		
52	15	.9004	.9148	56	65	.8595	1.128	61	25	1.035	1.095	65	75	.9258	1.543		
	20	.8951	.9204		70	.8493	1.146		30	1.023	1.109		80	.9152	1.481		
	25	.8884	.9275		75	.8408	1.162		35	1.010	1.125		90	.9063	1.506		
	30	.8805	.9361		80	.8344	1.174		40	.9946	1.144		66	0	1.152	1.152	
	35	.8716	.9462		90	.8290	1.185		45	.9787	1.165			10	1.146	1.158	
	40	.8620	.9575		57	0	.9948		.9948	50	.9623			1.188	15	1.139	1.165
	45	.8518	.9701			10	.9908		.9989	55	.9459			1.213	20	1.129	1.176
	50	.8414	.9835			15	.9858		1.004	60	.9299			1.239	25	1.116	1.190
	55	.8311	.9976			20	.9789		1.011	65	.9149			1.266	30	1.101	1.207
	60	.8212	1.012			25	.9703		1.020	70	.9015			1.292	35	1.084	1.227
65	.8120	1.026	30	.9602		1.032	75	.8903	1.316	40	1.066	1.250					
70	.8039	1.039	35	.9488		1.045	80	.8817	1.335	45	1.046	1.277					
75	.7972	1.050	40	.9363		1.060	90	.8746	1.352	50	1.026	1.308					
80	.7922	1.059	45	.9232		1.077	62	0	1.082	1.082	55	1.005	1.341				
90	.7880	1.066	50	.9096		1.095		10	1.077	1.087	60	.9850	1.377				
53	0	.9250	.9250	55	.8961	1.115		15	1.071	1.094	65	.9659	1.415				
	10	.9217	.9284	60	.8831	1.135		20	1.062	1.103	70	.9487	1.454				
	15	.9175	.9326	65	.8709	1.155		25	1.052	1.014	75	.9341	1.490				
	20	.9119	.9385	70	.8601	1.174		30	1.039	1.128	80	.9230	1.520				
	25	.9048	.9460	75	.8511	1.191		35	1.025	1.145	90	.9135	1.549				
	30	.8965	.9551	80	.8443	1.205		40	1.009	1.165	67	0	1.169	1.169			
	35	.8872	.9658	90	.8387	1.217		45	.9924	1.187		10	1.163	1.176			
	40	.8770	.9778	58	0	1.012		1.012	50	.9752		1.211	15	1.156	1.183		
	45	.8663	.9912		10	1.008	1.017	55	.9580	1.238		20	1.145	1.194			
	50	.8553	1.005		15	1.003	1.022	60	.9412	1.266		25	1.132	1.209			
55	.8444	1.021	20		.9956	1.029	65	.9254	1.295	30		1.117	1.226				
60	.8339	1.036	25		.9866	1.039	70	.9133	1.323	35		1.099	1.247				
65	.8241	1.051	30		.9760	1.051	75	.9005	1.349	40		1.080	1.272				
70	.8155	1.065	35		.9641	1.065	80	.8829	1.370	45		1.060	1.300				
75	.8084	1.077	40		.9510	1.081	90	.8829	1.389	50		1.038	1.332				
80	.8031	1.087	45		.9372	1.099	63	0	1.100	1.100	55	1.017	1.368				
90	.7986	1.095	50		.9230	1.118		10	1.094	1.105	60	.9956	1.406				
54	0	.9425	.9425	55	.9088	1.139		15	1.088	1.112	65	.9756	1.447				
	10	.9389	.9460	60	.8950	1.160		20	1.079	1.121	70	.9576	1.488				
	15	.9346	.9505	65	.8822	1.182		25	1.068	1.133	75	.9422	1.527				
	20	.9287	.9567	70	.8707	1.203		30	1.055	1.148	80	.9305	1.561				
	25	.9212	.9646	75	.8612	1.221		35	1.040	1.166	90	.9205	1.592				
	30	.9125	.9742	80	.8540	1.236		40	1.023	1.186	68	0	1.187	1.187			
	35	.9026	.9855	90	.8480	1.249		45	1.006	1.209		10	1.180	1.193			
	40	.8919	.9982	59	0	1.030		1.030	50	.9880		1.235	15	1.173	1.201		
	45	.8806	1.012		10	1.025	1.034	55	.9700	1.263		20	1.162	1.213			
	50	.8690	1.028		15	1.020	1.040	60	.9524	1.293		25	1.148	1.227			
55	.8575	1.044	20		1.012	1.048	65	.9358	1.324	30		1.132	1.246				
60	.8464	1.060	25		1.003	1.058	70	.9210	1.355	35		1.114	1.268				
65	.8361	1.076	30		.9918	1.070	75	.9085	1.383	40		1.094	1.294				
70	.8270	1.092	35		.9793	1.085	80	.8990	1.406	45		1.073	1.323				
75	.8194	1.105	40		.9656	1.102	90	.8910	1.427	50		1.051	1.357				
80	.8137	1.115	45		.9511	1.120	64	0	1.117	1.117	55	1.028	1.394				
90	.8090	1.124	50		.9362	1.141		10	1.112	1.123	60	1.006	1.435				
55	0	.9599	.9599	55	.9213	1.163		15	1.105	1.129	65	.9852	1.479				
	10	.9562	.9637	60	.9068	1.186		20	1.096	1.139	70	.9662	1.523				
	15	.9517	.9683	65	.8932	1.210		25	1.084	1.152	75	.9501	1.566				
	20	.9454	.9748	70	.8812	1.232		30	1.070	1.167	80	.9377	1.603				
	25	.9376	.9832	75	.8711	1.252		35	1.055	1.186	90	.9272	1.638				
	30	.9284	.9933	80	.8635	1.268		40	1.038	1.207	69	0	1.204	1.204			
	35	.9181	1.005	90	.8572	1.283		45	1.019	1.232		10	1.198	1.211			
	40	.9068	1.019	60	0	1.047		1.047	50	1.001		1.259	15	1.190	1.219		
	45	.8949	1.034		10	1.043	1.052	55	.9818	1.289		20	1.178	1.231			
	50	.8827	1.050		15	1.037	1.058	60	.9634	1.321		25	1.164	1.246			
55	.8705	1.067	20		1.029	1.066	65	.9460	1.354	30		1.148	1.266				
60	.8588	1.085	25		1.019	1.077	70	.9304	1.387	35		1.129	1.288				
65	.8479	1.102	30		1.008	1.090	75	.9173	1.418	40		1.108	1.315				
70	.8382	1.119	35		.9945	1.105	80	.9072	1.443	45		1.086	1.346				
75	.8302	1.133	40		.9811	1.123	90	.8988	1.466	50		1.063	1.382				
80	.8242	1.144	45		.9650	1.142	65	0	1.134	1.134	55	1.039	1.421				
90	.8192	1.154	50		.9493	1.164		10	1.129	1.140	60	1.016	1.465				
56	0	.9774	.9774	55	.9336	1.188		15	1.122	1.147	65	.9946	1.511				
	10	.9735	.9813	60	.9184	1.213		20	1.112	1.158	70	.9747	1.559				
	15	.9687	.9862	65	.9042	1.238		25	1.100	1.171	75	.9578	1.606				
	20	.9622	.9930	70	.8914	1.262		30	1.086	1.187	80	.9447	1.647				
	25	.9540	1.002	75	.8808	1.284		35	1.070	1.206	90	.9336	1.686				
	30	.9443	1.013	80	.8728	1.301		40	1.052	1.229	70	0	1.222	1.222			
	35	.9335	1.025	90	.8660	1.317		45	1.033	1.254		10	1.215	1.229			
	40	.9216	1.039	61	0	1.065		1.065	50	1.013		1.283	15	1.206	1.237		
	45	.9091	1.055		10	1.060	1.070	55	.9936	1.315		20	1.195	1.250			
	50	.8962	1.072		15	1.054	1.076	60	.9743	1.349		25	1.180	1.265			
55	.8834	1.091	20		1.046	1.084	65	.9561	1.384	30		1.163	1.285				
60	.8710	1.110					70	.9397	1.420								

TABLE 5—Continued

$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$	$\phi$	arcsin $k$	$E$	$F$
70	35	1.144	1.309	74	80	.9759	1.891	79	35	1.275	1.498	83	45	1.264	1.682
	40	1.122	1.337		90	.9613	1.962		40	1.245	1.537		50	1.227	1.746
	45	1.099	1.370	75	0	1.309	1.309		45	1.214	1.584		55	1.188	1.823
	50	1.075	1.407		10	1.301	1.317		50	1.181	1.639		60	1.150	1.914
	55	1.051	1.448		15	1.291	1.327		55	1.147	1.704		65	1.112	2.023
	60	1.027	1.494		20	1.277	1.342		60	1.113	1.779		70	1.076	2.154
	65	1.004	1.544		25	1.260	1.361		65	1.080	1.866		75	1.044	2.311
	70	.9830	1.596		30	1.240	1.385		70	1.050	1.966		80	1.017	2.496
	75	.9652	1.647		35	1.217	1.413		75	1.023	2.078		85	.9992	2.691
	80	.9514	1.692		40	1.191	1.448		80	1.001	2.195		90	.9925	2.794
90	.9397	1.735	45		1.163	1.488	90	.9816	2.340	84	0	1.466	1.466		
71	0	1.239	1.239		50	1.135	1.535	80	0		1.396	1.396	5	1.463	1.469
	10	1.232	1.246	55	1.105	1.588	5		1.394		1.399	10	1.456	1.477	
	15	1.223	1.255	60	1.076	1.649	10		1.387		1.406	15	1.443	1.490	
	20	1.211	1.268	65	1.048	1.718	15		1.376		1.418	20	1.425	1.509	
	25	1.196	1.285	70	1.022	1.793	20		1.360		1.434	25	1.403	1.533	
	30	1.179	1.305	75	.9992	1.871	25		1.340		1.457	30	1.377	1.565	
	35	1.158	1.330	80	.9814	1.947	30		1.316		1.485	35	1.346	1.604	
	40	1.136	1.359	90	.9659	2.028	35		1.289		1.519	40	1.313	1.650	
	45	1.112	1.393	76	0	1.326	1.326		40		1.259	1.560	45	1.276	1.706
	50	1.087	1.432		10	1.318	1.335		45	1.227	1.608	50	1.238	1.773	
55	1.062	1.476	15		1.308	1.345	50	1.193	1.666	55	1.198	1.853			
60	1.037	1.525	20		1.294	1.360	55	1.158	1.733	60	1.158	1.948			
65	1.013	1.578	25		1.276	1.380	60	1.122	1.813	65	1.119	2.063			
70	.9911	1.634	30		1.255	1.405	65	1.088	1.905	70	1.082	2.202			
75	.9724	1.689	35		1.231	1.434	70	1.056	2.012	75	1.049	2.373			
80	.9579	1.739	40		1.205	1.470	75	1.028	2.134	80	1.021	2.581			
90	.9455	1.788	45		1.176	1.512	80	1.055	2.265	85	1.022	2.814			
72	0	1.257	1.257		50	1.146	1.561	85	.9902	2.384	90	.9945	2.949		
	10	1.249	1.264	55	1.116	1.617	90	.9848	2.436	85	0	1.484	1.484		
	15	1.240	1.273	60	1.085	1.681	81	0	1.414		1.414	5	1.481	1.486	
	20	1.228	1.286	65	1.056	1.754		5	1.411		1.416	10	1.473	1.494	
	25	1.212	1.304	70	1.029	1.835		10	1.404		1.423	15	1.460	1.508	
	30	1.194	1.325	75	1.005	1.921		15	1.392		1.436	20	1.442	1.527	
	35	1.173	1.351	80	.9867	2.005		20	1.376		1.453	25	1.419	1.553	
	40	1.150	1.381	90	.9703	2.097		25	1.356		1.476	30	1.392	1.585	
	45	1.125	1.417	77	0	1.344		1.344	30		1.331	1.505	35	1.361	1.625
	50	1.099	1.457		10	1.335		1.353	35		1.303	1.540	40	1.326	1.673
55	1.073	1.504	15		1.325	1.363		40	1.272		1.582	45	1.289	1.731	
60	1.047	1.555	20		1.310	1.379		45	1.239	1.633	50	1.249	1.800		
65	1.022	1.612	25		1.292	1.399	50	1.204	1.693	55	1.208	1.883			
70	.9990	1.672	30		1.270	1.424	55	1.168	1.763	60	1.167	1.983			
75	.9794	1.733	35		1.246	1.455	60	1.132	1.846	65	1.127	2.103			
80	.9642	1.788	40		1.218	1.492	65	1.096	1.944	70	1.088	2.252			
90	.9511	1.843	45		1.189	1.536	70	1.063	2.058	75	1.053	2.437			
73	0	1.274	1.274		50	1.158	1.587	75	1.034	2.191	80	1.024	2.669		
	10	1.267	1.282	55	1.126	1.646	80	1.010	2.339	85	1.004	2.949			
	15	1.257	1.291	60	1.095	1.714	85	.9935	2.477	90	.9962	3.131			
	20	1.244	1.305	65	1.064	1.791	90	.9877	2.542	86	0	1.501	1.501		
	25	1.228	1.323	70	1.036	1.878	82	0	1.431		1.431	5	1.498	1.504	
	30	1.209	1.345	75	1.011	1.972		5	1.429		1.434	10	1.490	1.512	
	35	1.188	1.372	80	.9917	2.065		10	1.421		1.441	15	1.477	1.526	
	40	1.164	1.403	90	.9744	2.172		15	1.409		1.454	20	1.458	1.546	
	45	1.138	1.440	78	0	1.361		1.361	20		1.393	1.472	25	1.435	1.572
	50	1.111	1.483		10	1.353		1.370	25		1.371	1.495	30	1.407	1.605
55	1.084	1.532	15		1.342	1.381		30	1.346		1.525	35	1.375	1.646	
60	1.057	1.586	20		1.327	1.397		35	1.318		1.561	40	1.340	1.696	
65	1.031	1.647	25		1.308	1.418		40	1.286		1.605	45	1.301	1.755	
70	1.007	1.711	30		1.286	1.444		45	1.252	1.657	50	1.261	1.827		
75	.9862	1.777	35		1.260	1.476	50	1.215	1.719	55	1.219	1.913			
80	.9702	1.838	40		1.232	1.515	55	1.178	1.793	60	1.176	2.017			
90	.9563	1.901	45		1.201	1.560	60	1.141	1.880	65	1.134	2.144			
74	0	1.292	1.292		50	1.170	1.613	65	1.104	1.983	70	1.094	2.302		
	10	1.284	1.299	55	1.137	1.675	70	1.069	2.106	75	1.058	2.501			
	15	1.274	1.309	60	1.104	1.746	75	1.039	2.250	80	1.028	2.761			
	20	1.261	1.323	65	1.072	1.828	80	1.014	2.416	85	1.006	3.098			
	25	1.244	1.342	70	1.043	1.922	85	.9965	2.580	90	.9976	3.355			
	30	1.225	1.365	75	1.017	2.024	90	.9903	2.660	87	0	1.518	1.518		
	35	1.202	1.392	80	.9965	2.129	83	0	1.449		1.449	5	1.516	1.521	
	40	1.177	1.425	90	.9781	2.253		5	1.446		1.451	10	1.507	1.530	
	45	1.151	1.464	79	0	1.379		1.379	10		1.439	1.459	15	1.494	1.544
	50	1.123	1.509		10	1.370		1.388	15		1.426	1.472	20	1.475	1.564
55	1.094	1.560	15		1.359	1.399		20	1.409		1.490	25	1.451	1.591	
60	1.066	1.618	20		1.343	1.416		25	1.387		1.514	30	1.422	1.625	
65	1.039	1.682	25		1.324	1.437		30	1.362		1.545	35	1.389	1.667	
70	1.014	1.752	30		1.301	1.465		35	1.232		1.582	40	1.353	1.718	
75	.9928	1.824						40	1.299		1.628	45	1.314	1.780	

TABLE 5—Concluded

$\phi$	$\arcsin k$	$E$	$F$	$\phi$	$\arcsin k$	$E$	$F$	$\phi$	$\arcsin k$	$E$	$F$	$\phi$	$\arcsin k$	$E$	$F$
87	50	1.272	1.854	89	30	1.452	1.666	90	22	1.514	1.631	90	58	1.230	2.105
	55	1.229	1.944		35	1.418	1.710		23	1.509	1.637		59	1.221	2.130
	60	1.185	2.052		40	1.380	1.764		24	1.504	1.643		60	1.211	2.157
	65	1.142	2.185		45	1.338	1.829		25	1.498	1.649		61	1.202	2.184
	70	1.100	2.352		50	1.294	1.908		26	1.492	1.656		62	1.192	2.213
	75	1.063	2.567		55	1.249	2.004		27	1.486	1.663		63	1.183	2.244
	80	1.031	2.856		60	1.202	2.122		28	1.480	1.670		64	1.173	2.275
	85	1.008	3.262		65	1.156	2.267		29	1.474	1.678		65	1.164	2.309
	90	.9986	3.643		70	1.112	2.454		30	1.467	1.686		66	1.155	2.344
					75	1.072	2.701		31	1.461	1.694		67	1.145	2.381
88	0	1.536	1.536	90	80	1.037	3.053	32	1.454	1.703	68	1.136	2.420		
	5	1.533	1.539		85	1.011	3.633	33	1.447	1.712	69	1.127	2.461		
	10	1.525	1.547		90	.9998	4.741	34	1.440	1.721	70	1.118	2.505		
	15	1.510	1.562		0	1.571	1.571	35	1.432	1.731	71	1.110	2.551		
	20	1.491	1.583		1	1.571	1.571	36	1.425	1.741	72	1.101	2.600		
	25	1.466	1.610		2	1.570	1.571	37	1.417	1.752	73	1.093	2.652		
	30	1.437	1.645		3	1.570	1.571	38	1.409	1.763	74	1.084	2.708		
	35	1.404	1.689		4	1.570	1.572	39	1.401	1.775	75	1.076	2.768		
	40	1.366	1.741		5	1.569	1.573	40	1.393	1.787	76	1.069	2.832		
	45	1.326	1.805		6	1.568	1.574	41	1.385	1.799	77	1.061	2.903		
50	1.283	1.881	7	1.566	1.575	42	1.377	1.812	78	1.054	2.979				
55	1.239	1.974	8	1.565	1.577	43	1.368	1.826	79	1.047	3.062				
60	1.194	2.087	9	1.563	1.578	44	1.359	1.840	80	1.040	3.153				
65	1.149	2.226	10	1.561	1.581	45	1.351	1.854	81	1.034	3.255				
70	1.106	2.403	11	1.559	1.583	46	1.342	1.869	82	1.028	3.370				
75	1.067	2.634	12	1.556	1.585	47	1.333	1.885	83	1.022	3.500				
80	1.034	2.954	13	1.554	1.588	48	1.324	1.901	84	1.017	3.652				
85	1.010	3.441	14	1.551	1.591	49	1.315	1.918	85	1.013	3.832				
90	.9994	4.048	15	1.548	1.595	50	1.306	1.936	86	1.009	4.053				
			16	1.541	1.598	51	1.296	1.954	87	1.005	4.339				
89	0	1.553	1.553	17	1.541	1.602	52	1.287	1.973	88	1.003	4.743			
	5	1.550	1.556	18	1.537	1.606	53	1.278	1.993	89	1.001	5.435			
	10	1.542	1.565	19	1.533	1.610	54	1.268	2.013	90	1.000	$\infty$			
	15	1.527	1.580	20	1.528	1.615	55	1.259	2.035						
	20	1.507	1.601	21	1.524	1.620	56	1.249	2.057						
	25	1.482	1.630		1.519	1.625	57	1.240	2.080						

dropping out the terms containing the shearing stresses  $\tau_{\rho\theta}$  and  $\tau_{\theta z}$  which are equal to zero because of the symmetry. Computations made with this equation show that, in the special case when Poisson's ratio is equal to 0.5,  $\sigma_\theta$  is the intermediate principal stress  $\sigma_2$  and that, therefore, the two roots of the equation are actually the major and minor principal stresses.

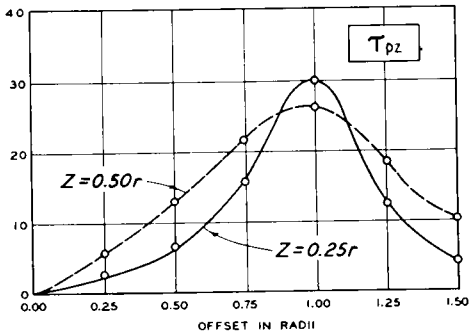
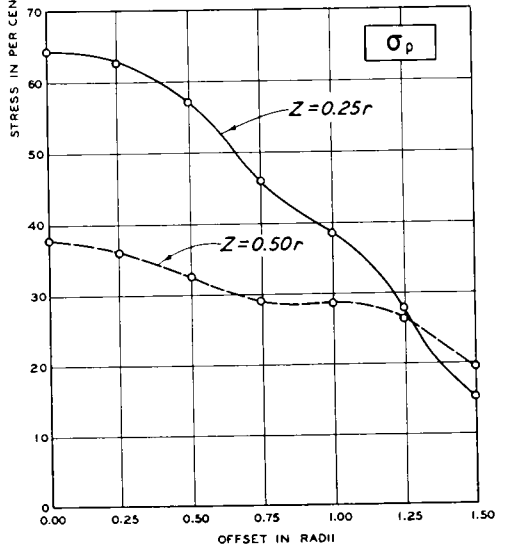
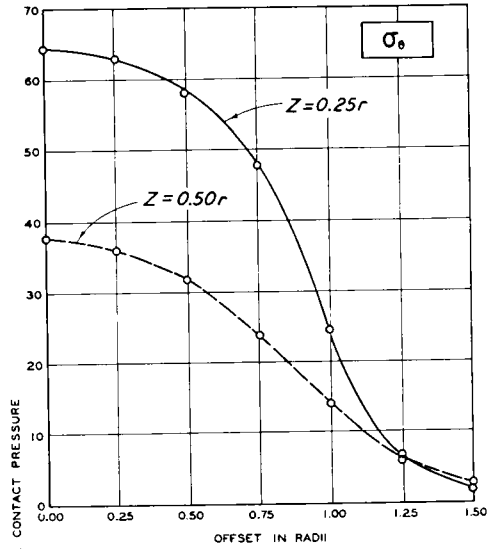
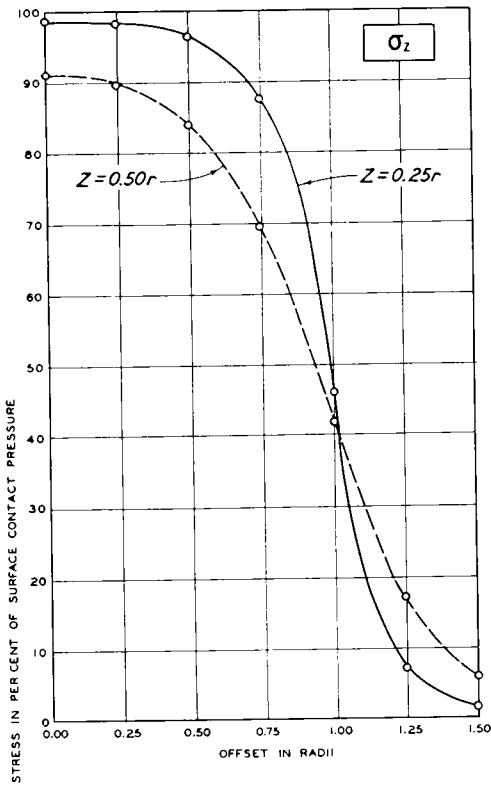
Expressions for the direct computation of these stresses and also for the maximum shearing stress which is equal to half their difference were obtained by substituting the terms shown on lines 24, 26, and 31 of table 3 for  $\sigma_2$ ,  $\sigma_\rho$ , and  $\tau_{\rho z}$ . These expressions are shown on lines 28, 29, and 30 of table 3. The formulas also make it possible to derive an expression for the tangent of the angle of the major principal plane. This angle is designated as  $\beta$  and is the angle formed by the major principal plane with the horizontal. It is measured clockwise from the horizontal which puts the first quadrant on the lower right for all positive values of the offset distance  $\rho$ .

In the special case when Poisson's ratio is 0.5, the tangent of  $2\beta$  can be computed from the expression shown on line 32 of table 3. It

is to be noted that this expression does not contain any elliptic integrals. It can also be shown in this case that the angle  $\beta$  is equal to  $180 - \frac{1}{2}(\theta - \gamma)$  and that the major principal plane is, therefore, perpendicular to the bisector of the angle  $v$  (see figures, table 2).

*Summation of forces over an area.* One of the problems encountered in the measurement of stresses within an earth mass is that of cell accuracy. To determine whether the cells measuring normal and shearing stresses are subject to any consistent over- or under-registration, a certain section of the earth was considered as a free body and the total forces acting over the body's several faces summed up. To check the accuracy of the values and the validity of the summation methods, the forces were also summed up theoretically by integration of the appropriate formulas. Three such integrations are shown on lines 40, 41, and 42 of table 3.

*Numerical examples.* To illustrate the use of the formulas, numerical examples are presented in which values for the stresses, strains, and deflections are computed at two points within the solid. The numerical values for the several elliptic integrals appearing in the



NOTE. LINES ARE COMPUTED STRESS.  
POINTS REPRESENT STRESSES READ FROM  
NEWMARK'S CHARTS.

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Figure 1. Computed values of coordinate stresses.

formulas are obtained from tables compiled by Legendre (8) in 1826. These tables have been photographically reproduced and are appended to the report listed as reference 1.

The values of  $E$ ,  $E'$ ,  $K$ ,  $K'$ ,  $E_{(k, \phi)}$ , and  $F_{(k, \phi)}$  are read from Legendre's tables by taking  $\phi$  (shown in the left column of each page) as the upper limit of integration and the value of  $k$  or  $k'$  as the sine of the angle shown at the head of each column in parentheses after the letters  $E$  or  $F$ . The complete integrals  $E$ ,  $E'$ ,  $K$ , and  $K'$  are thus read from the bottom line of every second page of Legendre's tables for the value of  $\phi$  of  $90^\circ$  and in the column indicated for arc sine  $k$  or  $k'$ .  $E$  and  $E'$  are read from the "E" column and  $K$  and  $K'$  from the "F" column. Where  $k$ ,  $k'$ , or  $\phi$  have values between those shown in the table, interpolations are made in the usual way.

For purposes of computation, it is convenient to note that arc sine  $k$  and arc sine  $k'$  are complementary angles and that where the angle  $v$  is greater than  $90^\circ$ , its cosine is negative. For those who do not need the accuracy of Legendre's tables, a brief abstract of the tables is appended hereto.

Values are computed at two points as follows:

<i>Example 1</i>	
$r$	1.0
$z$	0.5
$\rho$	0.5
$p$	100 psi
$E_m$	10,000 psi
$v$	0.5
$\tan \phi = \frac{z}{r - \rho}$	1.000
$\phi$	$45^\circ$
$\sin \phi$	0.7071
$R_1 = \frac{z}{\sin \phi}$	0.7071
$\tan \gamma = \frac{z}{r + \rho}$	0.3333
$\sin \gamma$	0.3162
$R_2 = \frac{z}{\sin \gamma}$	1.5811
$R_1 R_2$	1.1180
$\cos v$	-0.4472
$a$	0.8945
$b$	1.3417
$ab$	1.2000
$k = \frac{R_1}{R_2}$	0.4472
$\arcsin k$	$26.56^\circ$
$\arcsin k'$	$63.44^\circ$

$E'$	1.1785
$K'$	2.2575
$E_{(k, \phi)}$	1.0790
$F_{(k, \phi)}$	0.7709
$J$	0.8004
$B = 1 - \frac{J}{\pi}$	0.8767
$m = R_2 E'$	0.7210
$n = R_1 K'$	1.8633
$bm - n$	1.5963
$bn - m$	0.9037
$A = \frac{z}{2\pi\rho^2}$	0.2785
	0.3183

$\sigma_p =$	32.51 psi
$\sigma_2 = \sigma_\theta =$	31.64 psi
$\sigma_z =$	83.95 psi
$\sigma_{v=0.1} =$	148.11 psi
$\sigma_1 =$	87.00 psi
$\sigma_3 =$	29.47 psi
$\tau_{max} =$	28.76 psi
$\tau_{pz} =$	12.86 psi

$\epsilon_p =$	-0.002529 in/in
$\epsilon_\theta =$	-0.002659 in/in
$\epsilon_z =$	0.005188 in/in
$\omega_p =$	0.001330 in
$\omega_\theta =$	0.00 in
$\omega_z =$	0.012305 in

*Example 2*

$r$	1.0
$z$	0.5
$\rho$	2.0
$p$	100 psi
$E_m$	10,000 psi
$v$	0.5
$\tan \phi = \frac{z}{\rho - r}$	0.5000
$\phi$	$26.56^\circ$
$\sin \phi$	0.4472
$R_1 = \frac{z}{\sin \phi}$	1.1181
$\tan \gamma = \frac{z}{r + \rho}$	0.1667
$\sin \gamma$	0.1644
$R_2 = \frac{z}{\sin \gamma}$	3.0414
$R_1 R_2$	3.4006
$\cos v$	0.9557
$a$	-0.8087
$b$	1.5438
$ab$	-1.2485
$k = \frac{R_1}{R_2}$	0.3676
$\arcsin k$	$21.57^\circ$
$\arcsin k_1$	$68.43^\circ$
$E'$	1.1324
$K'$	2.4375

$K' - E'$	1.3051
$E_{(k,\rho)}$	0.4614
$F_{(k,\rho)}$	0.4657
$J$	0.5169
$B = 1 - \frac{J}{\pi}$	0.1645
$m = R_2 E'$	3.4441
$n = R_1 K'$	2.7254
$bm - n$	2.5916
$bn - m$	0.7634
$A = \frac{z}{2\pi\rho^2}$	0.0199

$\sigma_\rho = 9.41$ psi
$\sigma_2 = \sigma_\theta = 0.65$ psi
$\sigma_z = 1.05$ psi
$\sigma_{vol} = 11.07$ psi
$\sigma_1 = 10.37$ psi
$\sigma_2 = 0.05$ psi
$\tau_{max} = 5.16$ psi
$\tau_{\rho z} = 3.03$ psi
$\epsilon_\rho = 0.000853$ in/in
$\epsilon_\theta = -0.000456$ in/in
$\epsilon_z = -0.000397$ in/in
$\omega_\rho = 0.000912$ in
$\omega_\theta = 0.00$ in
$\omega_z = 0.004007$ in

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